Development of physical and mathematical models of the ducker to calculate the critical length of its sections between supports on the riverbed

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Abstract. The issues related to the construction and diagnosis of ducker crossings, signs of their operable condition are considered, with an emphasis on the development of a mathematical model for determining the critical length of the underwater pipeline spans between its supports in an unstable riverbed. A water-filled steel ducker of a certain diameter and wall thickness is subject to consideration for calculating the critical length, where a set of initial data necessary for design and construction is determined on the basis of initial technical, hydraulic and strength parameters. The results of the automated calculation are presented, which reflect the values of the following parameters: drag forces, Reynolds numbers, Froude criteria, pipeline weight, pushing and stretching forces, longitudinal stresses in the pipe wall from internal pressure and tensile force. On the basis of the obtained printouts of the results of the automated calculation, an analysis of the possible change in the value of the critical length of the ducker transition between the supports in the case of correction or clarification of some parameters regarding changes in the flow rates of water in the river and the permissible value of the deflection boom of the pipeline between the supports was carried out.

Keywords: ducker, construction technologies, condition diagnostics, mathematical model, critical length

Introduction
Performing work on the installation of a ducker in water bodies in an open manner on a riverbed is a complex of construction measures, including blocking the movement of water with a tongue or earth embankment, and with a large width or the impossibility of completely blocking the movement of water in the river, as a rule, using rope scraper installations, hydraulic monitors or dredgers. At the same time, one of the priority tasks when installing a ducker above the bottom of a watercourse with an unstable channel is the installation of supports supporting the ducker.

To immerse the pipeline thread to the bottom of the reservoir, the following technologies can be used [1-3]: dragging along the bottom, free immersion from floating supports; sequential build-up of pipe modules on the seabed or river bottom.

In order to prevent the pipeline from surfacing on the surface of the watercourse, it is necessary to perform ballasting operations with semi-rings of reinforced concrete, weighting

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saddle-shaped or annular single loads and anchor devices [4-7]. After completion of work on the installation of the duckline, it is necessary to fill the structure with soil when using a barge or hydraulic monitors.

In conditions of unstable hydraulic regime of the reservoir, unsteady and prone to erosion of the soils forming the channel, in order to ensure effective and long-term operation of the ducker junction, the installation of supports under the pipeline with a certain distance relative to each other is not excluded [8-9]. This explains the need to solve this urgent problem of determining the critical lengths between the ducker supports.

With prolonged operation of the ducker, it is possible to change its technical condition, which can be classified as: serviceable; operable; marginal (inoperable).

With regard to the state of the channel under the ducker crossings, signs of serviceable condition include: the location of the ducker along the bottom of the water body in the ground, respectively, design and regulatory and technical documentation; the stable condition of the bottom of the water body, the presence of fortified banks; the absence of defects in the walls of the pipeline, welded joints and supports.

Signs of a working condition include: minor deformations of the pipeline body (dents, corrugations); the presence of minor defects and damage to the ducker that do not exceed permissible values and do not affect the process of transporting water; a minor violation of shore protection; the presence of not exceeding the minimum permissible residual wall thickness of the pipeline, etc.

In the extreme (inoperable) condition of the ducker, its further operation is unacceptable without carrying out repair and restoration work, or is impractical due to the complexity (high cost) of restoration.

As evidenced by the practice of construction and operation of duckers, when it comes to critical lengths between the supports of a steel underwater passage, in this case a number of factors are subject to consideration, which primarily include such as pipe diameters, as well as their wall thicknesses, determined using various technical diagnostic tools [10-12].

Modern methods of instrumental inspection of the ducker, including in-line diagnostics of pipelines using non-destructive testing methods (without violating their integrity) include, for example, magnetometric methods, flaw scanners that detect the type of defects on steel pipelines and record changes in their parameters over time, as well as other complexes that work taking into account the stage of degradation in various geographical zones and seasons of work [13-20].

When examining exposed or sagging areas, diving television systems or cameras installed on board remote-controlled underwater vehicles are used to provide underwater video shooting [21].

In the material below, an integrated approach to determining the critical lengths between the supports of the ducker junction is considered, where a significant number of factors influencing the choice of the desired distance between the supports ensuring reliable operation of the ducker in an unstable channel are analyzed.

**Materials and methods**

The research materials are underwater pressure pipelines (duckers) installed at the bottom of the watercourse on supports, and the method is the development of physical and mathematical models of the ducker operation and calculation of the critical length $L_{cr}$ of sections of the sagging part of a round steel ducker filled with water in the profile, which has a certain diameter and wall thickness, with appropriate horizons of high and intertidal waters above the arch of the pipe in the place of possible erosion of the bottom in the riverbed (Fig. 1).
The ultimate goal of the research presented below is to calculate the critical length of the ducker between the supports based on the analysis of the technical, hydraulic and strength characteristics of the object under consideration.

**Research results and their interpretation**

The following basic parameters of the ducker operation are considered as initial information for the development of physical and mathematical models: the value of the water velocity in the bottom layer; unstable soils in the channel with its possible erosion; the location of the pipeline at a certain height from the bottom of the channel, which exceeds the diameter of the ducker; insignificant wave formation (the wavelength is small compared to the specified depths of the watercourse); the use of anticorrosive waterproofing of the pipeline, for example, normal bitumen according to SNiP II-45-75 «Main pipelines» with a certain thickness of a layer of bitumen-rubber mastic and a protective wrapper made of fiberglass according to GOST 15836-79 Bitumen-rubber insulating mastic.

The essence and sequence of the problem to be solved are described below.

When liquid flows around underwater pipelines, they experience a force effect: drag and lift. At the same time, in conditions of insignificant wave formation on the river, the wave effect on the pipeline is not taken into account, and with a relatively large location of the ducker relative to the bottom of the river, the lifting force is not taken into account.

Thus, for the given conditions of the location of the ducker, the drag $P$ acting per unit length of the pipeline is determined by the formula:

$$ P = C_x \cdot \frac{v^2 \cdot \rho \cdot D_{ht}}{2}, $$

where $C_x$ - the drag coefficient (determined depending on the Reynolds number according to the graphs in Figure 2); $\rho$ - the density of river water; $v$ - the velocity of the river flow; $D_{ht}$ - the outer diameter of the pipeline.

The Reynolds number $Re$ for the flow of water flowing into the pipeline is determined by the formula:

$$ Re = \frac{v \cdot D_{ht}}{\nu} $$

where $\nu$ - the kinematic viscosity of the water in the river depending on the temperature of the water in the watercourse $t$.

Having the calculated values of the Reynolds number, the value of the drag coefficient $C_x$ is determined from the graph in Figure 2 using the corresponding curves (for example, under number 13).
In order to take into account the influence of the free surface of water in the river, the coefficient $C_x$ should be multiplied by the coefficient $K_1$, which takes into account the ratio of the horizon of high waters to the outer diameter of the ducker, i.e. $K_1 = h_1/D_h$, and the coefficient $K_2$, which takes into account the ratio of the horizon of intertidal waters to the outer diameter of the ducker, i.e. $K_2 = h_2/D_h$.

If the calculated value of $K_1 > 4$, then the correction of $K_1$ is not entered.

After that, the values of the Froude criterion $Fr$ are evaluated:

- for high water:
  \[ Fr_1 = \frac{v^2}{gH_1} \]

- for the autumn:
  \[ Fr_2 = \frac{v^2}{gH_2} \]

where $g$ - the acceleration of gravity; $H_1$ and $H_2$ - respectively, the distances to the bottom of the reservoir at high water and at low water.

In the case of obtaining the calculated value of the Froude numbers $Fr < 1$, the coefficient $K_2$ is not entered.

The next step of the algorithm is to determine the stability of the pipeline. Earlier it was noted that the pipeline is located at a certain height from the bottom of the riverbed, which exceeds the diameter of the ducker.

The calculation of the lifting force is performed according to the formula:

\[ \frac{s}{D_h}, \]

where $s$ - the distance from the bottom of the pipeline to the bottom of the river

If the lifting force exceeds 1, it is not subject to subsequent accounting.

The buoyant (Archimedean) force is calculated in case the pipeline is installed in place until it is filled. Excessive force (when the pipeline is full) leads to an increase in the stability of the pipeline.

Stability is understood as a condition of the pipeline in which it will be at rest in a predetermined position with the most unfavorable combination of loads seeking to remove the pipeline from this position.

To determine the stability of the pipeline, a number of preliminary calculation operations are carried out to identify the weight (mass):

- one meter of pipeline $Q$:
  \[ Q = m_v \cdot g \]

- water in one meter of pipeline $Q_w$:
  \[ Q_w = \rho \cdot g \cdot \pi \cdot (D_h^2 - 2l)^2 / 4 \]

- insulation on the pipeline $Q_{is}$:
\[ Q_{tr} = \rho_{is} \cdot \pi \cdot \frac{D_h^2 - D_i^2}{4} \]

- the pipeline as a whole, taking into account the insulation and the water filling the pipe \( Q_{tr} \):

\[ Q_{tr} = Q + Q_h + Q_{is}, \]

where \( m_{tr} \) - the mass of one meter of pipe; \( D_h \) - the diameter of the ducker (external without waterproofing), \( \rho_{is} \) - the density of the insulation material

The pushing (Archimedean) force \( A \) is determined by the formula:

\[ A = \rho \cdot g \cdot \pi \cdot \frac{D_h^2}{4} \]

The ring stresses \( \delta_{kc} \) in the pipe wall from the internal pressure are calculated using the formula:

\[ \delta_{kc} = k \cdot P_{bn} \cdot D_{bn}/(2 \cdot l) \]

The longitudinal stresses \( \delta_{pr} \) in the pipe wall from the internal pressure are determined by the formula:

\[ \delta_{pr} = \eta \cdot \delta_{kc} \]

The longitudinal tensile force \( H_{pr} \) is calculated using the formula:

\[ H_{pr} = 10^6 \cdot \delta_{pr} \cdot \pi \cdot D_h \cdot a \]

Weighing force \( q_h \):

\[ q_h = Q_{tr} - K \cdot P - A \]

Stiffness coefficient \( K_j \) of the pipeline section:

\[ K_j = E \cdot \left( \pi \cdot D_h^4/64 \right) \cdot \left[ 1 - \left( D_{bn}/D_h \right)^4 \right], \]

where \( l \) - the wall thickness of the pipeline; \( P_{bn} \) - the internal pressure in the pipeline, \( D_{bn} \) - the inner diameter of the ducker, \( K \) - the internal pressure overload coefficient, \( \eta \) - the transverse deformation coefficient (Poisson's ratio) for steel, \( a \) - the thickness of anticorrosive bitumen insulation, \( E \) - the rigidity of the steel from which the pipe is made.

The critical span length \( L_{kr} \) between the supports is determined by the results of solving the transcendental biquadratic equation:

\[ q_h = (40H_{pr} \cdot L_{kr}^2 + 384 \cdot K_j)/2 \cdot (5 \cdot L_{kr}^2), \]

where \( f \) - the magnitude of the deflection boom of the ducker section (permissible deflection when bending the pipe profile).

The mathematical model of the ducker operation presented above was the basis for the development of an automated program for calculating the critical span length of the sections of the ducker between the supports.

The initial information for the task under consideration is presented in Table 1.

<table>
<thead>
<tr>
<th>The name of the parameters and their dimension</th>
<th>The values of the parameters</th>
</tr>
</thead>
<tbody>
<tr>
<td>Diameter of the ducker (external without waterproofing) ( D_h ) (m)</td>
<td>0.8</td>
</tr>
<tr>
<td>Pipeline wall thickness ( l ) (m)</td>
<td>0.011</td>
</tr>
<tr>
<td>The pipeline is made of st3sp steel with rigidity ( E ) (MPa)</td>
<td>200000</td>
</tr>
<tr>
<td>Internal pressure in the pipeline ( P_{bn} ) (MPa)</td>
<td>1.177</td>
</tr>
<tr>
<td>Internal pressure overload factor ( K )</td>
<td>1.15</td>
</tr>
<tr>
<td>Coefficient of transverse deformation (Poisson's ratio) for steel ( \eta )</td>
<td>0.3</td>
</tr>
<tr>
<td>The magnitude of the deflection boom ( f )</td>
<td>0.03</td>
</tr>
<tr>
<td>The horizon of high waters ( h_1 ) (m)</td>
<td>10</td>
</tr>
<tr>
<td>The horizon of intertidal waters ( h_2 ) (m)</td>
<td>6</td>
</tr>
<tr>
<td>The velocity of water in the bottom layer ( v ) (m/s)</td>
<td>0.5</td>
</tr>
<tr>
<td>The distance of the pipeline from the bottom of the river ( s ) (m)</td>
<td>1.25</td>
</tr>
<tr>
<td>Anti-corrosion protection (bituminous) ( a ) (m)</td>
<td>0.008</td>
</tr>
<tr>
<td>Insulation material density ( \rho_{is} ) (kg/m³)</td>
<td>1250</td>
</tr>
</tbody>
</table>
Water temperature in the river $t (^{\circ}C)$ | 10
---|---
Kinematic viscosity of water in a river depending on temperature $\nu$ ($m^2/c$) | $1.306 \cdot 10^{-6}$
Drag coefficient $C_d$ | 0.9
Density of river water $\rho$ ($kg/m^3$) | 1000
The weight of one meter of pipe $m_n$ ($kg/m$) | 160.2

The results of the automated calculation are presented in Table 2.

<table>
<thead>
<tr>
<th>The name of the parameters and their dimension</th>
<th>The values of the parameters</th>
</tr>
</thead>
<tbody>
<tr>
<td>Diameter of the ducker with waterproofing $D_{ht}$ $(m)$</td>
<td>0.816</td>
</tr>
<tr>
<td>The inner diameter of the ducker $D_{bn}$ $(m)$</td>
<td>0.778</td>
</tr>
<tr>
<td>Drag force $P$ $(N/m)$</td>
<td>91.8</td>
</tr>
<tr>
<td>Reynolds number for the flow of water flowing into the pipeline $Re$</td>
<td>312404.29</td>
</tr>
<tr>
<td>The ratio for high water $h_1/D_{ht}$</td>
<td>12.25</td>
</tr>
<tr>
<td>The ratio for intertidal waters $h_2/D_{ht}$</td>
<td>7.35</td>
</tr>
<tr>
<td>The distance to the bottom of the reservoir at high water $H_1$ $(m)$</td>
<td>12.07</td>
</tr>
<tr>
<td>The distance to the bottom of the reservoir at the boundary $H_2$ $(m)$</td>
<td>8.07</td>
</tr>
<tr>
<td>Froude's criterion for high water $Fr_1$</td>
<td>0.002112</td>
</tr>
<tr>
<td>Froude's criterion for the boundary $Fr_2$</td>
<td>0.003159</td>
</tr>
<tr>
<td>Ratio $s/D_{ht}$</td>
<td>1.532</td>
</tr>
<tr>
<td>The weight of one meter of pipeline $Q$ $(N/m)$</td>
<td>1571.56</td>
</tr>
<tr>
<td>The weight of water in one meter of pipeline $Q_w$ $(N/m)$</td>
<td>4661.201</td>
</tr>
<tr>
<td>Weight of insulation on the pipeline $Q_{is}$ $(N/m)$</td>
<td>248.89</td>
</tr>
<tr>
<td>The weight of the pipe, taking into account the insulation and the water filling the pipe $Q_{tr}$ $(N/m)$</td>
<td>6481.65</td>
</tr>
<tr>
<td>The pushing (Archimedian) force $A$ $(N/m)$</td>
<td>5127.66</td>
</tr>
<tr>
<td>Annular stresses in the pipe wall from internal pressure $\delta_{k}$ (MPa)</td>
<td>47.87</td>
</tr>
<tr>
<td>Longitudinal stresses in the pipe wall from internal pressure $\delta_{pr}$ (MPa)</td>
<td>14.36</td>
</tr>
<tr>
<td>Longitudinal tensile force $H_{pr}$ $(N)$</td>
<td>288577.25</td>
</tr>
<tr>
<td>Weighing force $q_{h}$ $(N/m)$</td>
<td>1248.43</td>
</tr>
<tr>
<td>The coefficient of rigidity of the pipeline section $K_j$</td>
<td>755.52</td>
</tr>
<tr>
<td>Critical span length between supports $L_{kr}$ $(m)$</td>
<td>7.45</td>
</tr>
</tbody>
</table>

Analyzing the data in Table 2, it should be noted that the critical span length between the supports is 7.45 m.

If, for example, the total length of the ducker within the riverbed is 45 m, and the distance between the supports (in the amount of at least 6 spans) should not exceed 7.45 m, then it will be necessary to erect 5 supports without taking into account the attachment (pinching) of the ducker on both banks (see Figure 1).

With alternative options for changing the flow velocity of water in a watercourse, for example, 0.1 and 0.9 m/s, the critical lengths will be 7.17 and 8.27 m, respectively. Similar changes in the critical length between the supports can be traced with a different magnitude of the deflection boom.

If, with an arrow value of $f = 0.03$, the critical span length was $L_{kr} = 7.45 m$, then with a greater permissible curvature of $f = 0.05$, it will be $L_{kr} = 9.62 m$.

Thus, it can be stated that the critical length between the supports depends on a number of hydraulic parameters of the river flow and the assigned technical limitations.
Conclusions
1. Based on the creation of a physical model of the work of the ducker, analytical studies were carried out to develop a mathematical model of its operation when the pipeline is located above the bottom of a watercourse.
2. A general solution algorithm for determining the critical lengths of sections (spans) of the ducker between the supports is presented.
3. An automated calculation of the operation of the underwater ducker was performed with the determination of basic technical, hydraulic and strength parameters, the critical length between the supports and their number, which can be used by designers in solving such problems.
4. The analysis of the influence of some parameters on the change in the value of the critical span length of the ducker junction between the supports is carried out.

References


